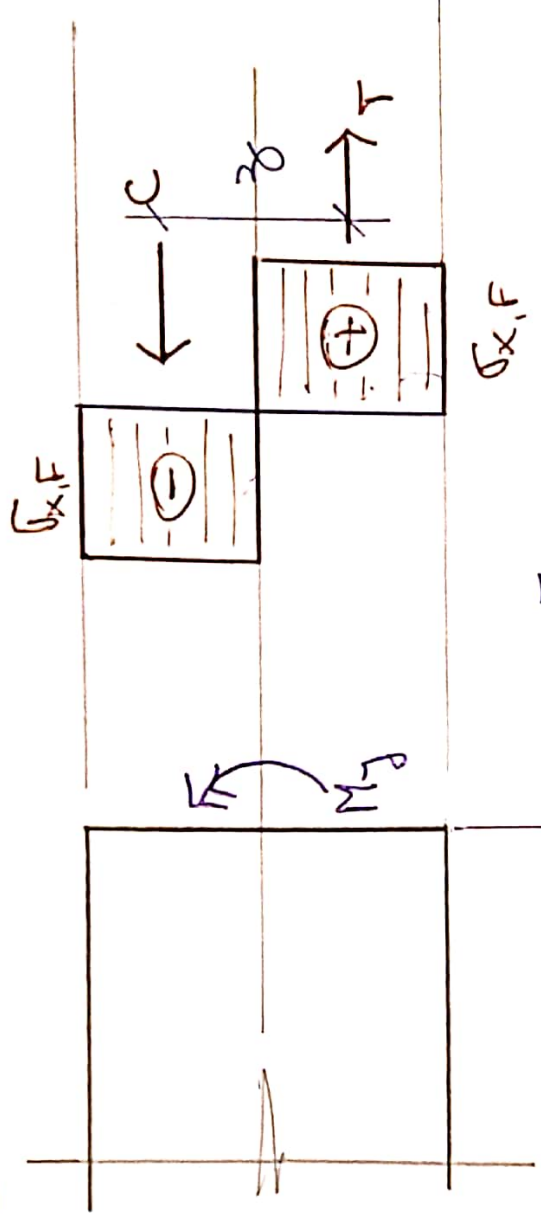


SECCIÓN TOTALMENTE PLÁSTIFICADA:



$$M_y = bh^2 \sigma_{x,F} - \frac{1}{3} b z^2 \sigma_{x,F}$$

$$M_y = bh^2 \sigma_{x,F} \left[1 - \frac{1}{3} \frac{z^2}{h^2} \right]$$

$$M_y = \frac{3}{2} M_p \left[1 - \frac{1}{3} \frac{z^2}{h^2} \right]$$

$$N = 0$$

$$M_p = T \cdot z = C \cdot z = \underbrace{\sigma_{x,F} \cdot b \cdot h}_{T=C} \cdot \underbrace{h}_{z} = bh^2 \sigma_{x,F}$$

$$T = \sigma_{x,F} \cdot b h = C$$

$$z = 2 \cdot \frac{h}{2} = h$$

$$M_y = M_p = bh^2 \sigma_{x,F} = \frac{3}{2} M_p \left[1 - \frac{1}{3} \frac{0}{h^2} \right] = \frac{3}{2} M_p$$

$$\rightarrow M_p = \frac{3}{2} M_p$$

• $\frac{M_P}{M_E} = \frac{3}{2} = k = \text{FACTOR DE FORMA,}$

$M_P = k M_E$

• $\frac{M_P}{\sigma_{x,F}} = \frac{bh^2 \sigma_{x,F}}{\sigma_{x,F}} = bh^2 = Z$

Z : MÓDULO PLÁSTICO DE LA SECCIÓN.

• M_P → MOMENTO DE PLASTIFICACIÓN
 ↳ DE TOTAL.
 ↳ MOMENTO ÚLTIMO O DE COLAPSO DE LA SECCIÓN.

• $M_P = Z \cdot \sigma_{x,F}$

▢ RECTANGULAR $k = 1,50$

▢ CUADRAS $k = 2,00$

Autor: Ing. Luis Nelson SOSTI $k \rightarrow 1,08$ y $1,14$.

• $M_E = \frac{2}{3} bh^2 \sigma_{x,F}$

$\frac{M_E}{\sigma_{x,F}} = \frac{\frac{2}{3} bh^2 \sigma_{x,F}}{\sigma_{x,F}} = \frac{2}{3} bh^2 = S$

→ MASH }
 → M_E }
 → M_P }
 S: MÓDULO ELÁSTICO DE LA SECCIÓN.

$I_y = \frac{b(2h)^3}{12} = \frac{8}{12} bh^3 = \frac{2}{3} bh^3$

$z_{max} = h$

$W_y = S_y = \frac{I_y}{z_{max}} = \frac{2}{3} bh^2$

$\frac{Z}{S} = \frac{bh^2}{\frac{2}{3} bh^2} = \frac{3}{2} = k$

$k = \frac{M_P}{M_E} = \frac{Z}{S}$

DEFORMACIONES:

• si $dx = 1 \rightarrow \frac{d\theta}{dx} = \kappa = \text{CURVATURA DE FLEXI\O{N}}$

$d\theta = \kappa$

$\epsilon_x = \frac{\Delta dx}{dx} \rightarrow \Delta dx = \epsilon_x$

$\Delta dx, z = z \cdot d\theta \rightarrow \epsilon_x = z \cdot \kappa$

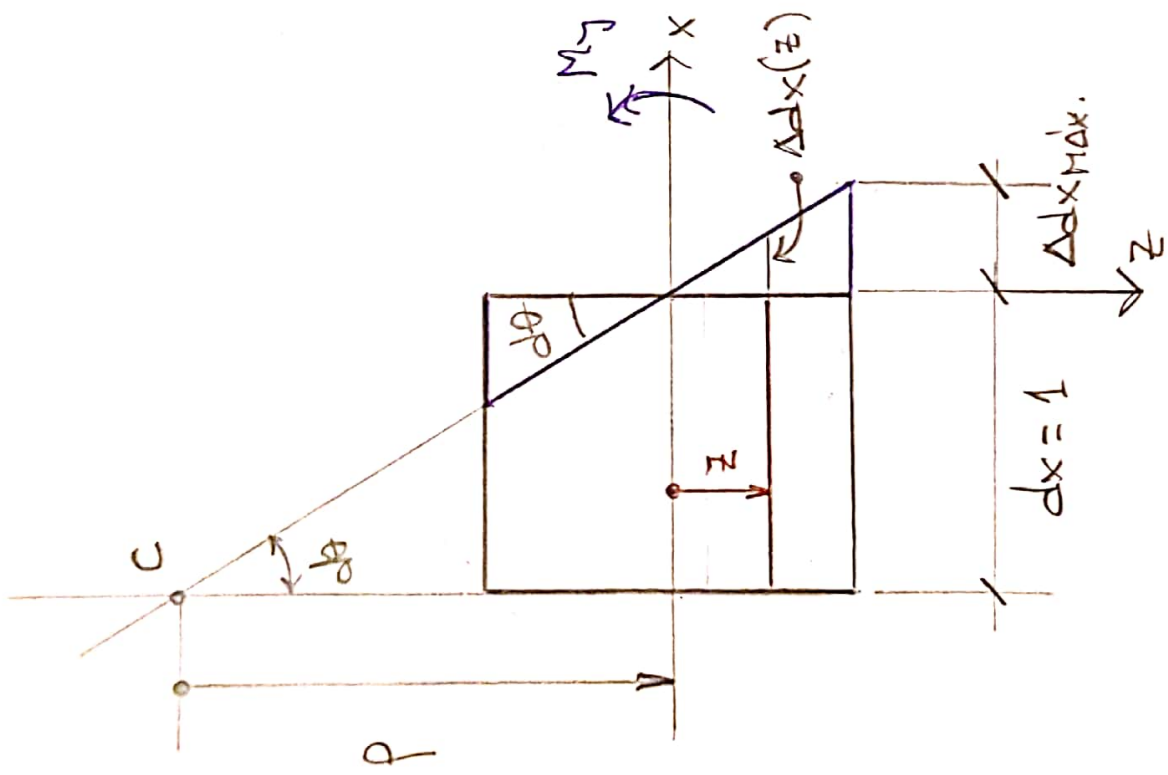
$\rho \cdot d\theta = dx \rightarrow \rho \cdot \kappa = 1 \rightarrow$

$\kappa = \frac{1}{\rho} ; \rho = \frac{1}{\kappa}$

$\epsilon_x = z \cdot \frac{1}{\rho}$

• si $z_E = h \rightarrow \epsilon_{x,F} = h \cdot \kappa_F \rightarrow \kappa_F = \frac{\epsilon_{x,F}}{h}$

$\epsilon_{x,F} = h \cdot \frac{1}{\rho_F} \rightarrow \rho_F = \frac{h}{\epsilon_{x,F}}$



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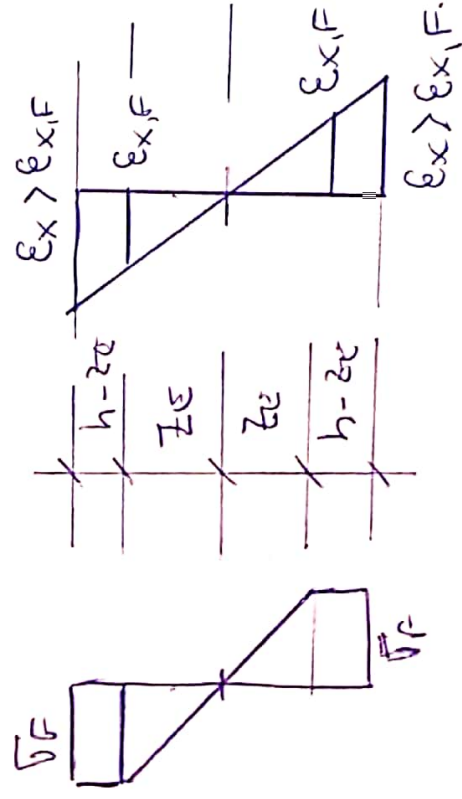
→ $h = \frac{\epsilon_{x,F}}{\alpha_F}$

(A)

$h = \rho_F \epsilon_{x,F}$

• si $z_E < h$:

$z = z_E \rightarrow \epsilon_x = \epsilon_{x,F}$



$z_E \cdot \alpha = \epsilon_{x,F} \rightarrow \alpha = \frac{\epsilon_{x,F}}{z_E}$

$z_E \cdot \rho = \epsilon_{x,F} \rightarrow \rho = \frac{z_E}{\epsilon_{x,F}}$

(B)
(A)

→ $\frac{z_E}{h} = \frac{\epsilon_{x,F}}{\alpha} \cdot \frac{\alpha_F}{\epsilon_{x,F}}$

$\frac{\alpha_F}{\alpha}$

→ $\frac{z_E}{h} = \frac{\rho \cdot \epsilon_{x,F}}{\rho_F \cdot \epsilon_{x,F}}$

$\frac{\rho}{\rho_F}$

REEMPLAZO EN LAS PRIMERAS 3 EXPRESIONES DE LA HOJA (1):

$M_y = \frac{3}{2} M_e \left[1 - \frac{1}{3} \frac{\alpha_F}{\alpha^2} \right]$

$M_y = \frac{3}{2} m_e \left[1 - \frac{1}{3} \frac{\rho^2}{\rho_F} \right]$

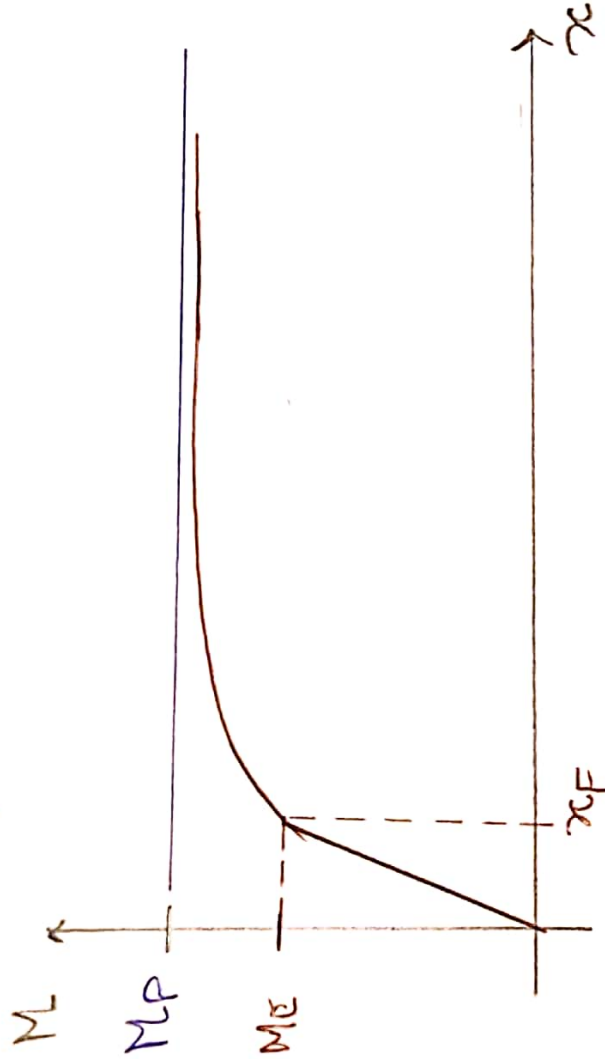
$z_E = \frac{\epsilon_{x,F}}{\alpha}$

(B)

$z_E = \rho \cdot \epsilon_{x,F}$

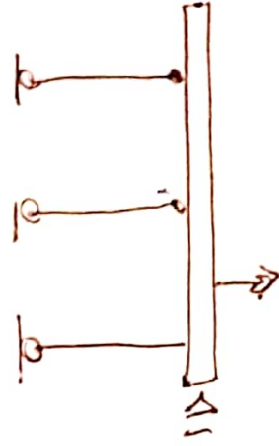
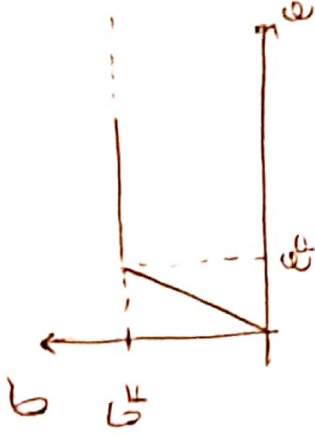
I) si $\epsilon_e = h \rightarrow x = x_F \wedge f = f_F$
(MOMENTO ELÁSTICO).

II) si $\epsilon_e = 0 \rightarrow x \rightarrow \infty \wedge f \rightarrow 0$.
(MOMENTO PLÁSTICO).



• si $M \leq M_E \rightarrow x = \frac{M y}{E I} \quad \epsilon = \frac{M y}{E I}$

$M y = E I \epsilon$



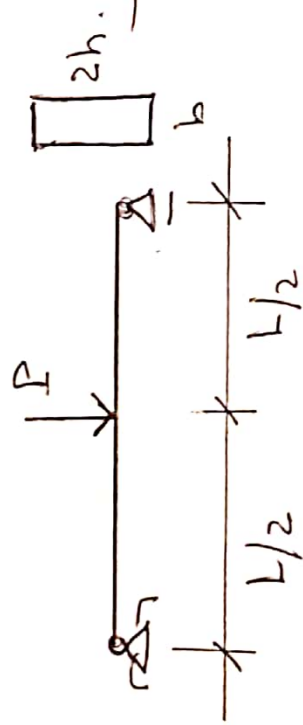
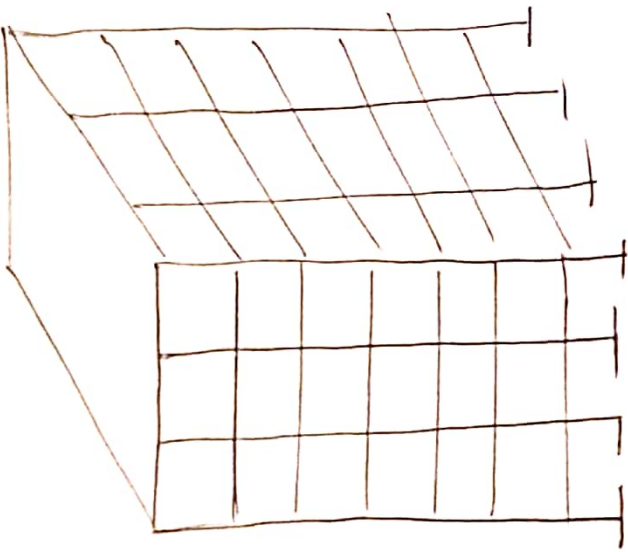
~~rotula~~
rotula
esquema
sente.

ROTULA O ARTICULACIÓN PLÁSTICA.

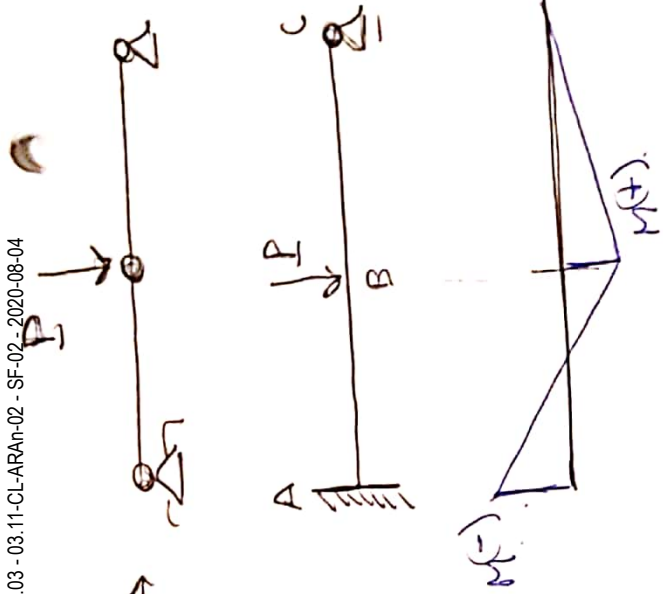
→ SE FORMA CUANDO

$M = M_P$

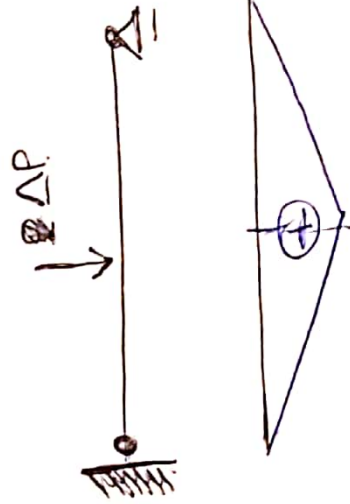
6/16



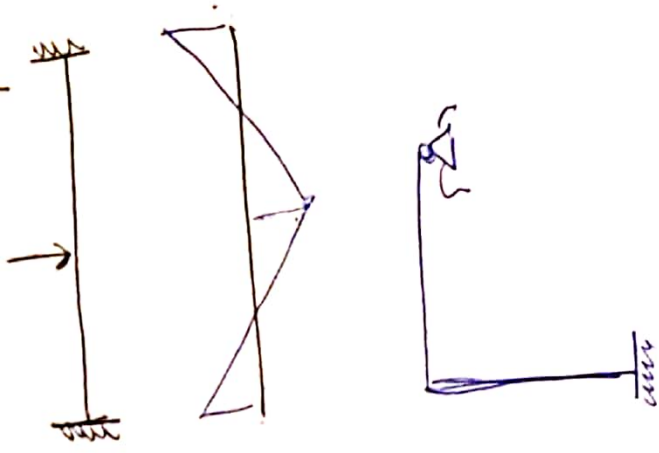
$P \text{ create } \rightarrow M_{y,adm}$
 $M_{y,e}$
 $M_{y,p}$



$M_A = M_P.$

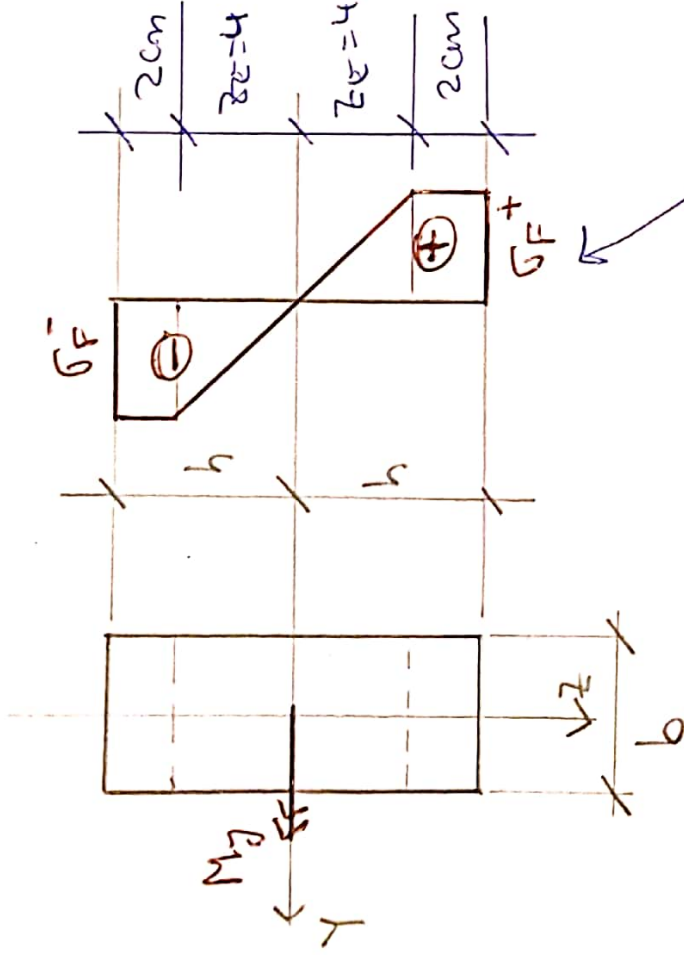


$P_C = P_u = P_j + \Delta P$



EXEMPLO:

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$$I_y = \frac{b(2h)^3}{12} = \frac{2}{3}bh^3 \quad z_{\max} = h.$$

$$\frac{I_y}{z_{\max}} = \frac{2}{3}bh^2 = \frac{2}{3}5 \cdot 6^2 = 120 \text{ cm}^3.$$

$$M_E = \frac{I_y}{z_{\max}} \cdot \sigma_F = 120 \text{ cm}^3 \cdot \frac{24 \text{ kN}}{\text{cm}^2}$$

$$M_E = 2880 \text{ kNcm} = 28,80 \text{ kNm}$$

$$M_P = \frac{3}{2} M_E = 1,50 \cdot 28,80 = 43,20 \text{ kNm}.$$

$M_E < M_y < M_P.$

DATOS:

$$b = 50 \text{ mm}.$$

$$h = 60 \text{ mm}$$

$$H = 120 \text{ mm}$$

$$\sigma_F^+ = \sigma_F^- = 24 \text{ kN/cm}^2$$

$$E = 20000 \text{ kN/cm}^2.$$

$$M_y = 28,8 \text{ kNm}.$$

$$M_y = \frac{3}{2} M_E \left[1 - \frac{1}{3} \frac{\sigma_F^2}{h^2} \right]$$

$$\frac{2}{3} \frac{M_y}{M_E} = 1 - \frac{1}{3} \frac{\sigma_F^2}{h^2}$$

$$\frac{1}{3} \frac{\sigma_F^2}{h^2} = 1 - \frac{2}{3} \frac{M_y}{M_E}$$

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$$Z_E = \sqrt{\left(1 - \frac{2}{3} \frac{M_E}{N_E}\right) 3h^2}$$

$$Z_E = \sqrt{\left(1 - \frac{2}{3} \frac{36,8}{28,8}\right) \cdot 3 \cdot 6^2}$$

$$Z_E = 4 \text{ cm}$$

CURVATURA Y RADIO DE CURVATURA:

$$\frac{Z_E}{h} = \frac{\chi_F}{\chi} \rightarrow \chi = \chi_F \frac{h}{Z_E}$$

$$E = 20000 \text{ kN/cm}^2$$

$$I_y = \frac{2}{3} b h^3 = \frac{2}{3} \cdot 5 \cdot 6^3$$

$$I_y = 720 \text{ cm}^4$$

$$h = 6 \text{ cm}$$

$$Z_E = 4 \text{ cm}$$

$$\chi_F = \frac{M_E}{E I_y}$$

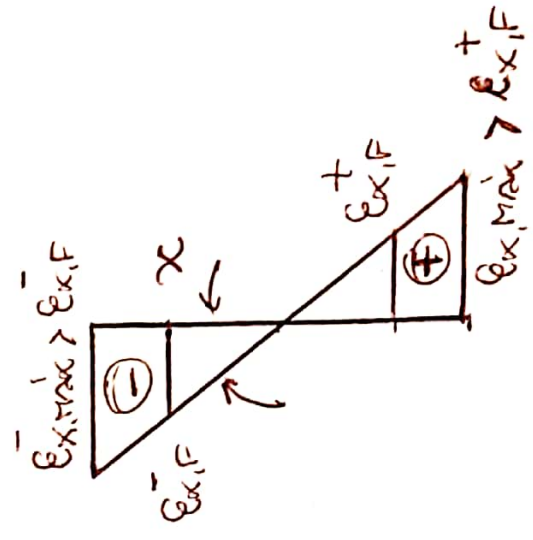
$$\chi_F = \frac{M_E}{E I_y} = \frac{2880 \text{ kN cm}}{20000 \frac{\text{kN}}{\text{cm}^2} \cdot 720 \text{ cm}^4}$$

$$\chi_F = 2 \cdot 10^{-4} \frac{1}{\text{cm}}$$

$$\rho = \frac{1}{\chi} = \frac{1}{2 \cdot 10^{-4}} = 5000 \text{ cm}$$

$$\rho = \frac{1}{\chi} = 3333 \text{ cm}$$

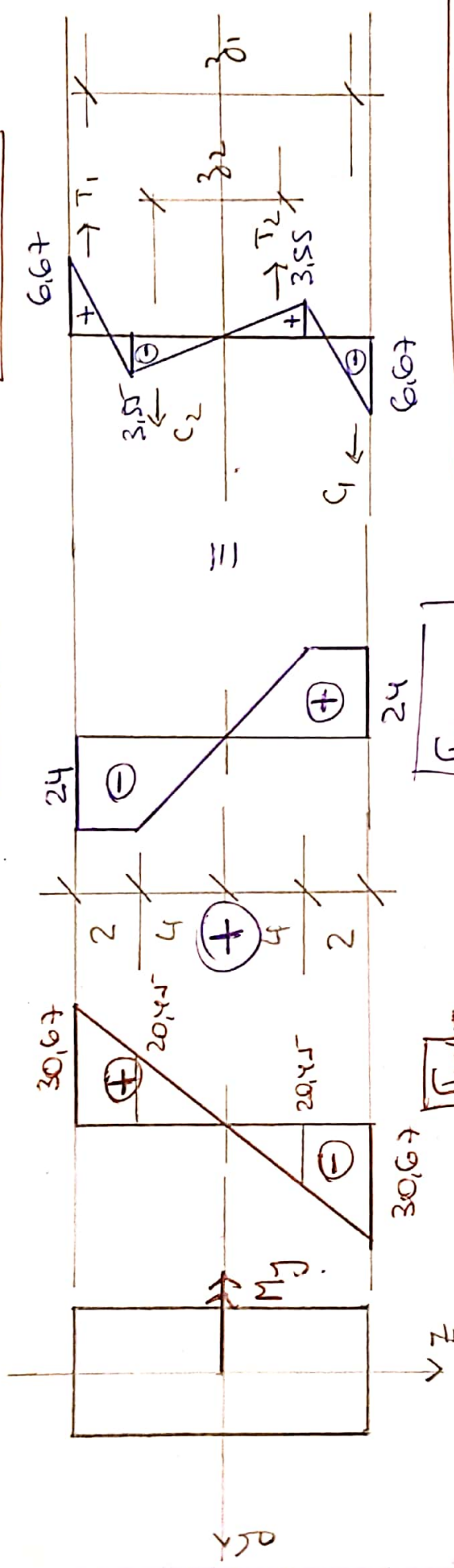
$$\rho = \frac{1}{\chi} = 3333 \text{ cm}$$



DESCARGA:

TENSIONES
RESIDUALES

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$M_y = 36,8 \text{ kNm}$

$$\sigma_{x, \text{desc}} = \frac{M_y}{I_y} \cdot z_{\text{máx}} = \frac{-36,8 \text{ kNm} \cdot 6 \text{ m}}{720 \text{ cm}^4}$$

$\sigma_{x, \text{desc}} = 30,67 \text{ kN/cm}^2$

$$\frac{30,67}{6} = \frac{\sigma_{x, z=4}}{4}$$

$$T_1 \cdot z_1 = C_1 \cdot z_1$$

$$T_2 \cdot z_2 = C_2 \cdot z_2$$

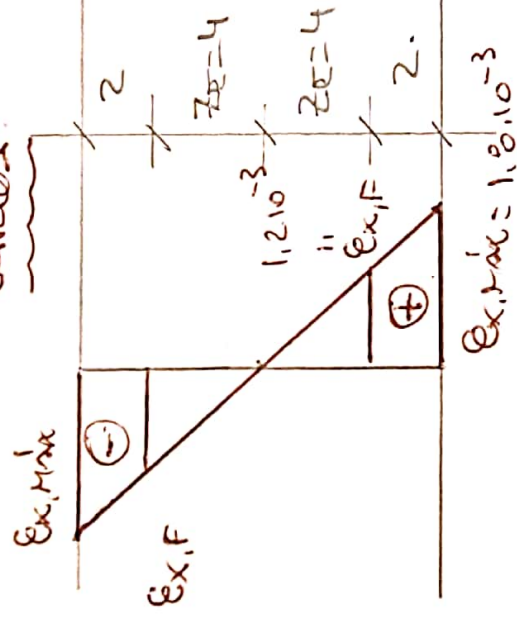
$|T_1 \cdot z_1| = |T_2 \cdot z_2|$

$|C_1 \cdot z_1| = |C_2 \cdot z_1|$

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DEFORMACIONES

CARGA:

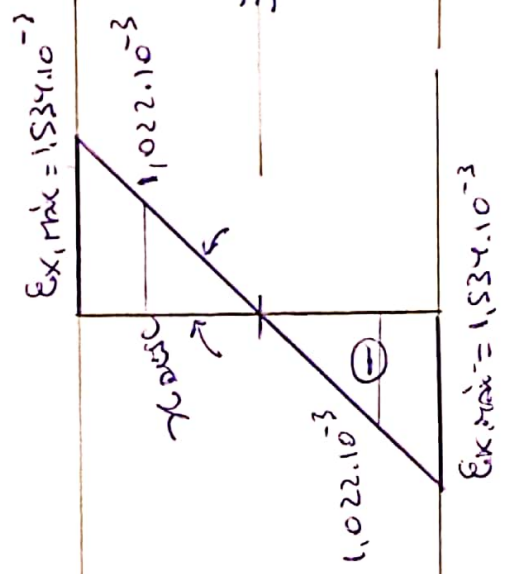


$$Ex, F = \frac{\sigma_{x, F}}{E} = \frac{24 \text{ kN} / \text{cm}^2}{20000 \text{ kg/cm}^2}$$

$$Ex, F = 1,2 \cdot 10^{-3} = 0,12 \%$$

$$\frac{Ex, \text{max}}{E} = \frac{Ex, F}{4} \rightarrow Ex, \text{max} = 1,8 \cdot 10^{-3}$$

DESPLAZA:



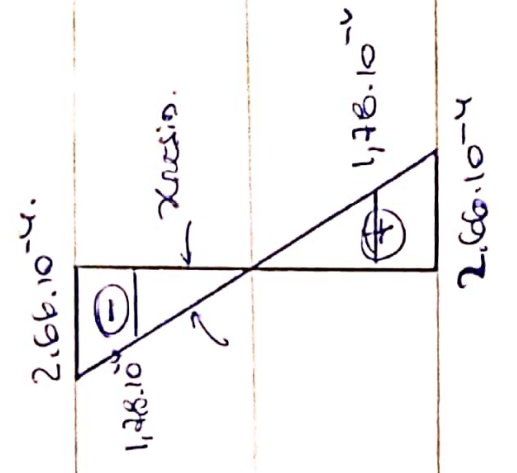
$$Ex, \text{max} = \frac{\sigma_{x, \text{max}}}{E}$$

$$Ex, \text{max} = \frac{30,62}{20000} = 1,534 \cdot 10^{-3}$$

$$\frac{Ex, \text{max}}{E} = \frac{Ex, \text{BE}}{4}$$

$$Ex, \text{BE} = 1,022 \cdot 10^{-3}$$

PRESI DEFORM:



$\frac{\sigma_{x, \text{res, BE}}}{E} = Ex, \text{res, BE}$
 $ACA - z_E \leq z \leq z_E$
 ES MÁXIMA LA VEY DE TROCAR.

$$\frac{3,57 \text{ MN}}{20000 \text{ kg/cm}^2} =$$

$$= 1,78 \cdot 10^{-4}$$

CURVATURA + RAYOS DE CURVATURA: $\chi_F = 2 \cdot 10^{-4} / \text{cm}$. $\rho_F = 5000 \text{ cm}$.

RESIDUAL

CARGA: $\chi = 3 \cdot 10^{-4} \frac{1}{\text{cm}}$

$\rho = 3333 \frac{1}{\text{cm}}$

$\chi_{\text{RESID}} = \frac{1,78 \cdot 10^{-4}}{4 \text{ cm}} = 4,45 \cdot 10^{-5} \frac{1}{\text{cm}}$

$\chi_{\text{RESID}} = \frac{2,66 \cdot 10^{-4}}{6 \text{ cm}} = 4,44 \cdot 10^{-5} \frac{1}{\text{cm}}$

DESCARGA:

$\chi_{\text{DESC}} = \frac{E_{\text{max, DESC}}}{h} = \frac{1,534 \cdot 10^{-3}}{6 \text{ cm}}$

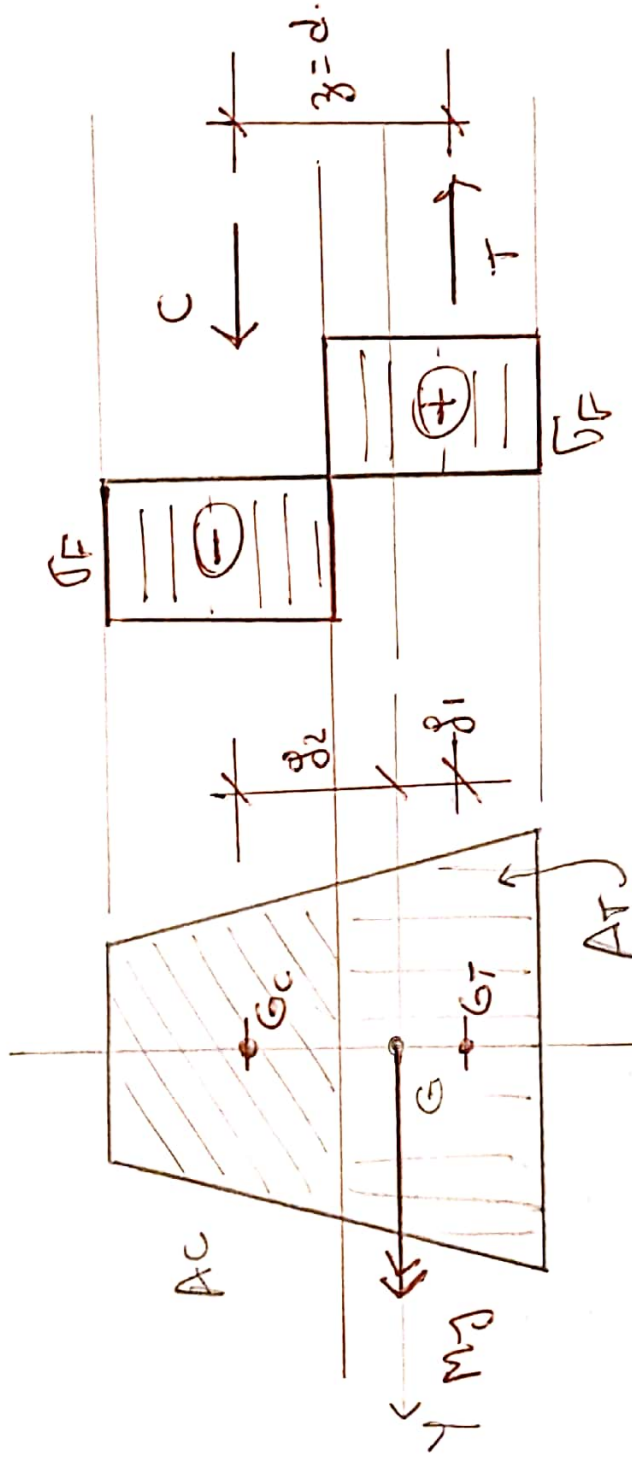
$\chi_{\text{DESC}} = 2,557 \cdot 10^{-4} \frac{1}{\text{cm}}$

$\rho_{\text{DESC}} = \frac{1}{\chi_{\text{DESC}}} = 3911 \frac{1}{\text{cm}}$

$\rho_{\text{RESID}} = 22471 \text{ cm}$

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SECCIONES CON 1 EJE DE SIMETRÍA: → SECCIÓN TOTALMENTE PUNIFORME.



$$T = C$$

$$\int \sigma_F A_T = \int \sigma_F A_C$$

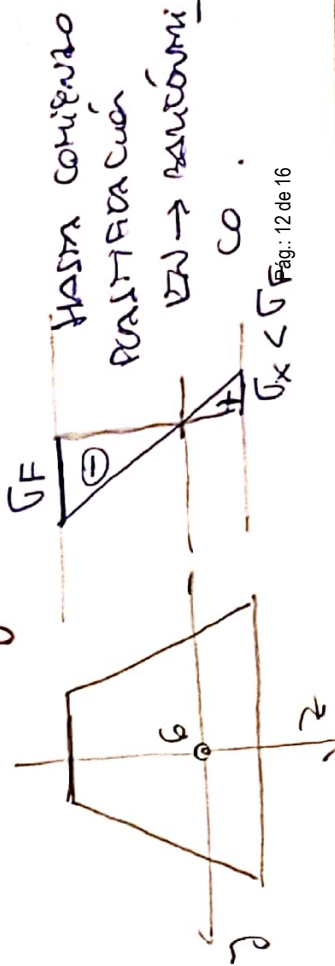
$$A_T = A_C$$

• Si $A = \text{ÁREA DE LA SECCIÓN}$

$$A_T = A_C = \frac{A}{2}$$

EL EJE NEUTRO DIVIDE A LA SECCIÓN TRANSVERSAL EN 2 ÁREAS IGUALES.

$$C \cdot z = T \cdot z = M_y = \int P \cdot z$$



HASTA COMIENZO DE LA SECCIÓN TRANSVERSAL EN → TRANSFORMACIÓN

$$N = 0 = \int \sigma_x \cdot dA \rightarrow T = C = R$$

$$M_y = \int \sigma_x \cdot z \cdot dA$$

RESULTANTE TRACCION = $T = \int \sigma_F \cdot A_T$

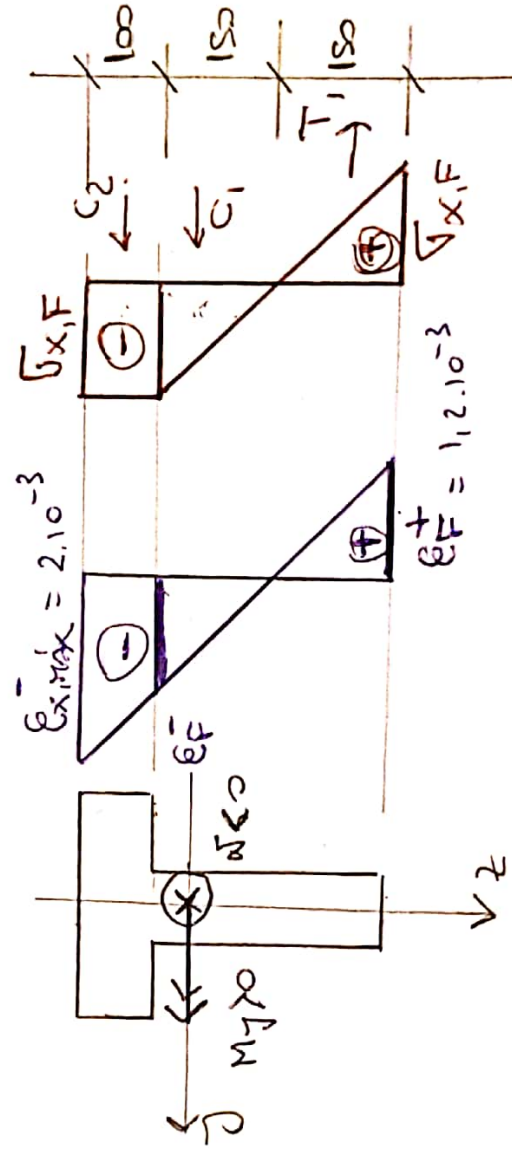
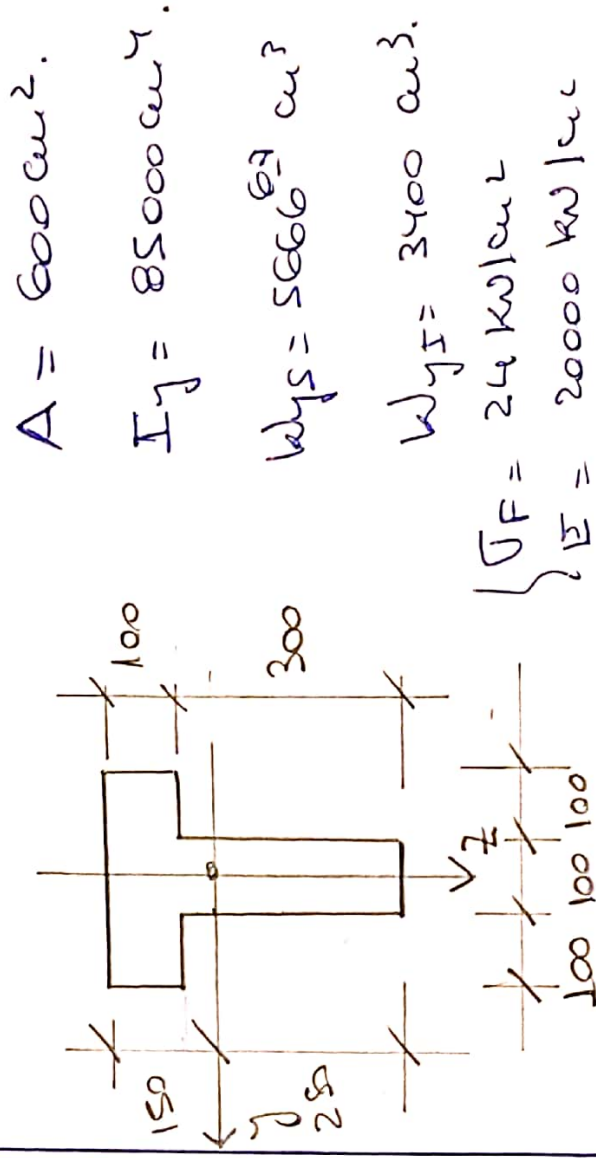
COMPRESION = $C = \int \sigma_F \cdot A_C$

FLEXIÓN COMPUESTA:

- ES SIMILAR AL PROBLEMA DE LA FLEXIÓN SIMPLE → EL INCENTIVO DE LOS ESFUERZOS INTERNOS → HACE QUE $\sigma_x > \sigma_{x,F}$.
- EL PROBLEMA ES MÁS COMPLEJO PORQUE → SON 2 ~~LOS~~ LOS ESF. INTERNOS QUE PRODUCEN TENSIONES.

$$\left. \begin{aligned} N &= \int_A \sigma_x \, dA \\ M_y &= \int_A \sigma_x \cdot z \cdot dA \end{aligned} \right\}$$

EJEMPLO:



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$$N = \underbrace{(24) \cdot \frac{10 \cdot 15}{2}}_{C_1} - \underbrace{(24) \cdot \frac{10 \cdot 15}{2}}_{C_2} = \boxed{-7200 \text{ kN}}$$

$$M_y = \underbrace{24 \cdot \frac{10 \cdot 15}{2} \cdot (25-5)}_{M_{y,T_1}} + \underbrace{24 \cdot \frac{10 \cdot 15}{2} \cdot (15-15)}_{M_{y,C_1}} + \underbrace{24 \cdot 10 \cdot 30 \cdot (15-r)}_{M_{y,C_2}} = 36000 + 0 + 72000$$

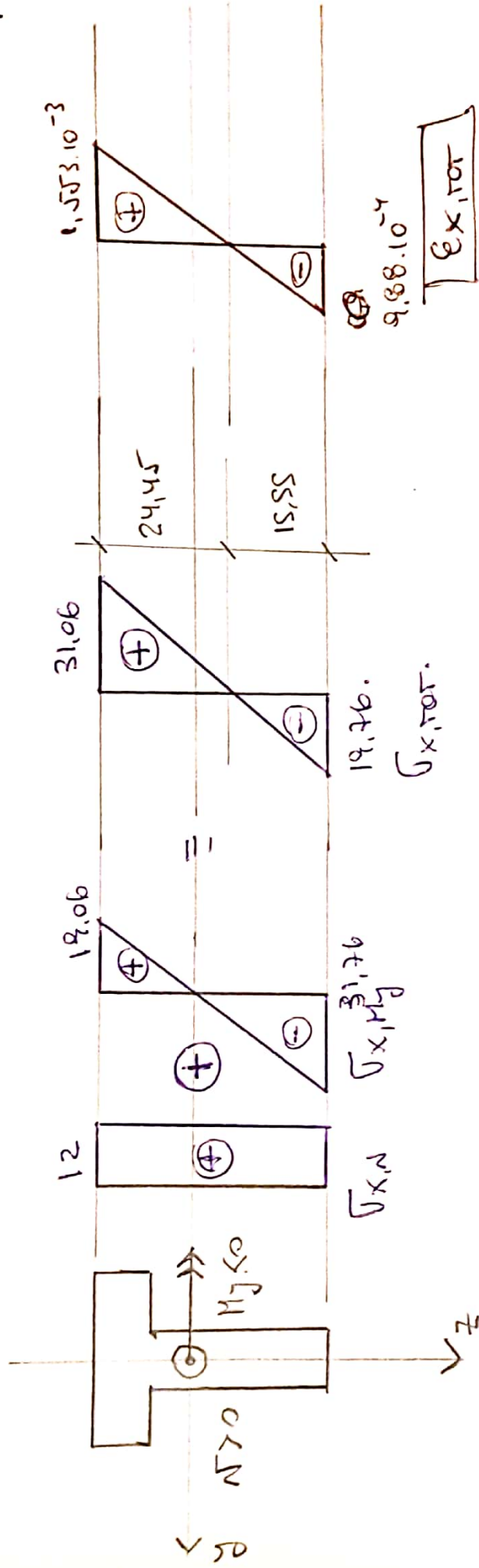
REDUCCION A G DE LA SECCION.
RESPECTO AL BARRICENTRO DE LA SECCION.

$$M_y = 108.000 \text{ kNm} = \boxed{1080 \text{ kNm}}$$

FLEXION COMPUESTA RECTA.

$$|\epsilon_F^+| = |\epsilon_F^-| = \frac{\sigma_F}{E} = \frac{24}{20000} = 1,2 \cdot 10^{-3}$$

$$\frac{\epsilon_{x,\max}^-}{25} = \frac{\epsilon_F^-}{15} \rightarrow \epsilon_{x,\max}^- = 2,0 \cdot 10^{-3}$$



$$\sigma_{x,N} = \frac{N}{A} = \frac{7200 \text{ kN}}{6000 \text{ cm}^2} = 12 \text{ kN/cm}^2$$

$$\sigma_{x,M_y}^2 = \frac{M_y}{W_{yI}} = \frac{108000}{566667} = 19,06 \frac{\text{kN}}{\text{cm}^2}$$

$$\sigma_{x,M_y}^I = \frac{M_y}{W_{yI}} = \frac{108000}{34000} = 31,76 \frac{\text{kN}}{\text{cm}^2}$$

$$\epsilon_{x,S} = \frac{\sigma_{x,S,TOT}}{E} = \frac{31,06}{20000} = 1,553 \cdot 10^{-3}$$

$$\epsilon_{x,I} = \frac{\sigma_{x,I,TOT}}{E} = \frac{19,76}{20000} = 9,88 \cdot 10^{-4}$$

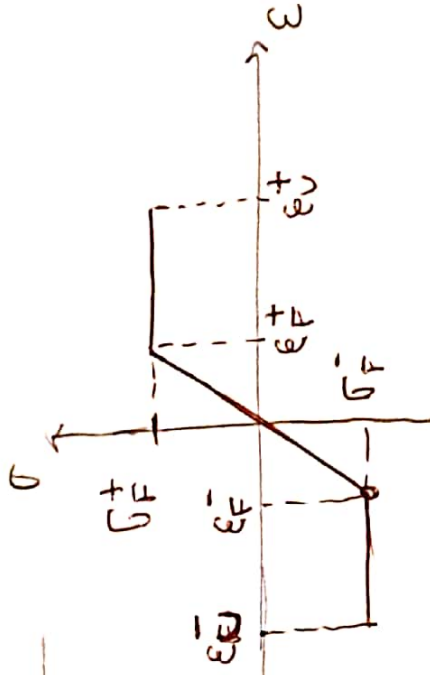
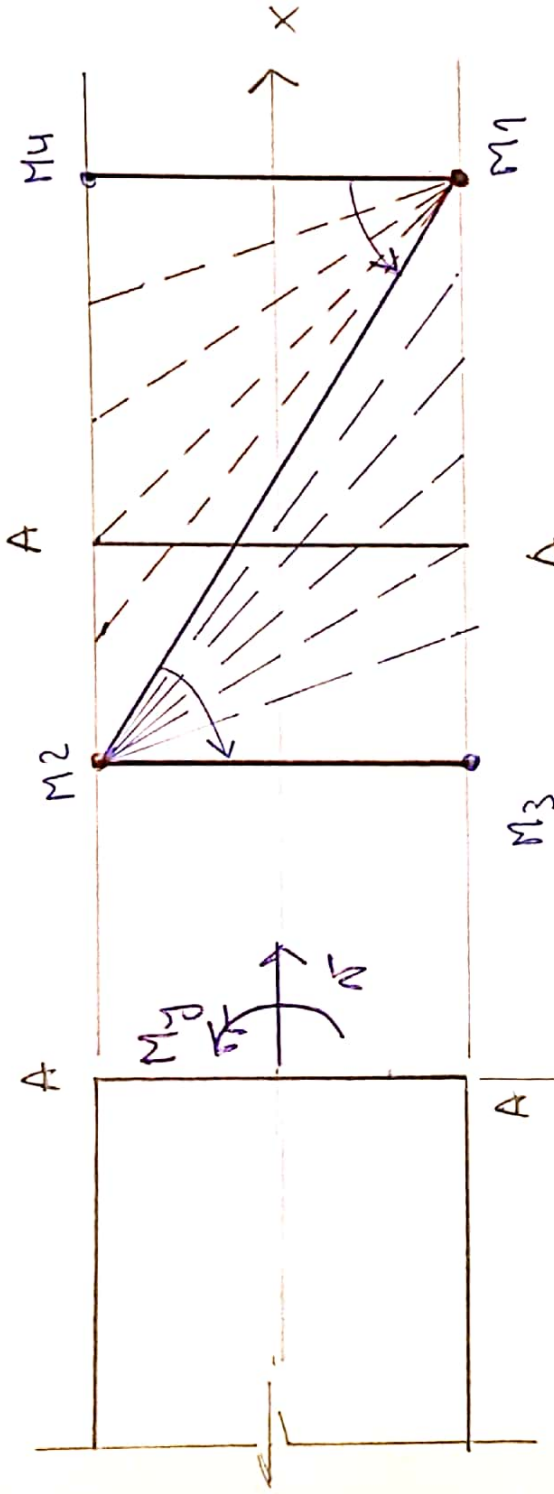
$$\kappa_{DESC} = \frac{1,553 \cdot 10^{-3}}{24,45 \text{ cm}} = 6,352 \cdot 10^{-5} \frac{1}{\text{cm}}$$



→ HACER LA PARTE RESIDUAL

Mi = Ptos de notación.

PLANOS LÍMITES DE DEFORMACIÓN.



$\epsilon_U^+ = \epsilon_U^c$ (5%)
 $\epsilon_U^- = \epsilon_U^c$ (3%)

DADES

$\epsilon_U^+ \rightarrow$ FIBRAS INF.

UBICACIÓN EN

- TORS. • TORS Y DEF. NO SIMULTANEAS.
- N Y M.
- DEFORMAC.

