

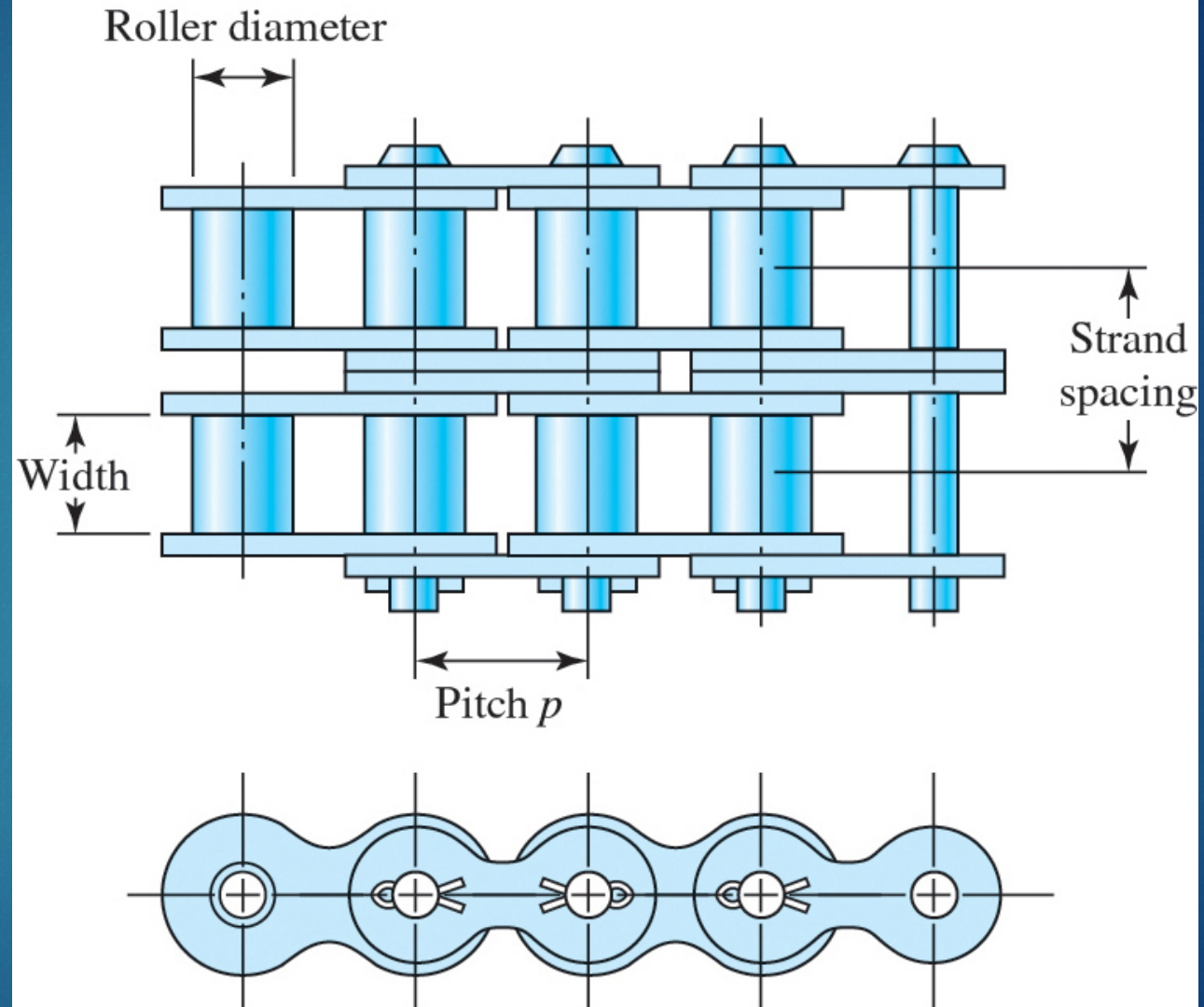


67.29 Proyecto de Máquinas

TRANSMISIÓN POR CADENAS

Cadena de rodillos

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► Fig.
17-16

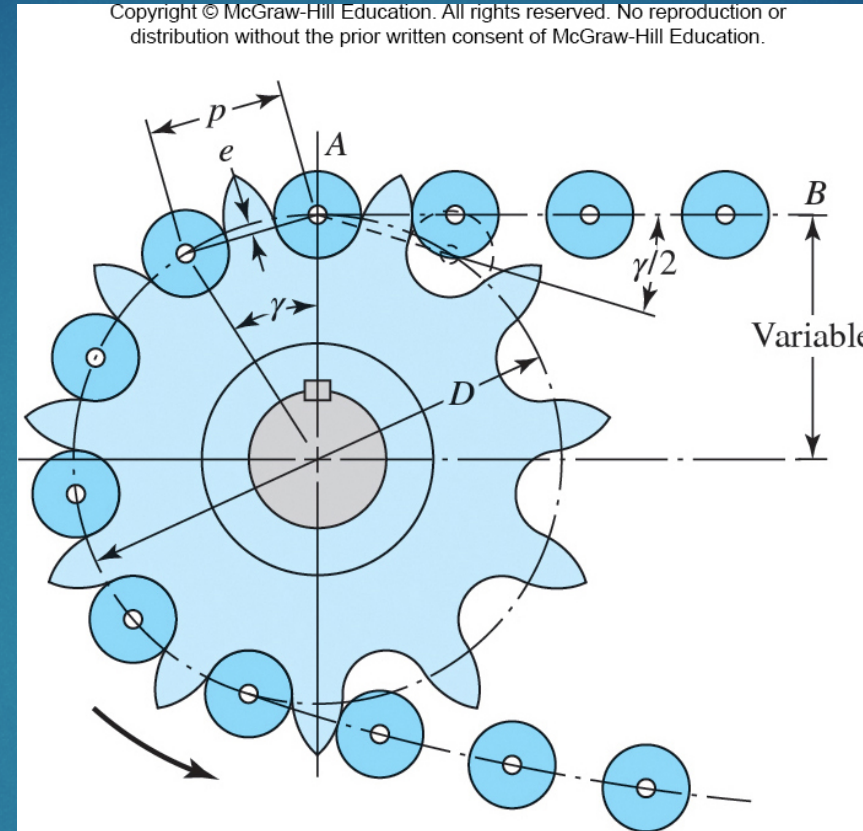
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Dimensiones de las cadenas de rodillos estándar estadounidenses

| ANSI Chain Number | Pitch, in (mm) | Width, in (mm) | Minimum Tensile Strength, lbf (N) | Average Weight, lbf/ft (N/m) | Roller Diameter, in (mm) | Multiple-Strand Spacing, in (mm) |
|-------------------|----------------|----------------|-----------------------------------|------------------------------|--------------------------|----------------------------------|
| 25 | 0.250 (6.35) | 0.125 (3.18) | 780 (3 470) | 0.09 (1.31) | 0.130 (3.30) | 0.252 (6.40) |
| 35 | 0.375 (9.52) | 0.188 (4.76) | 1 760 (7 830) | 0.21 (3.06) | 0.200 (5.08) | 0.399 (10.13) |
| 41 | 0.500 (12.70) | 0.25 (6.35) | 1 500 (6 670) | 0.25 (3.65) | 0.306 (7.77) | — |
| 40 | 0.500 (12.70) | 0.312 (7.94) | 3 130 (13 920) | 0.42 (6.13) | 0.312 (7.92) | 0.566 (14.38) |
| 50 | 0.625 (15.88) | 0.375 (9.52) | 4 880 (21 700) | 0.69 (10.1) | 0.400 (10.16) | 0.713 (18.11) |
| 60 | 0.750 (19.05) | 0.500 (12.7) | 7 030 (31 300) | 1.00 (14.6) | 0.469 (11.91) | 0.897 (22.78) |
| 80 | 1.000 (25.40) | 0.625 (15.88) | 12 500 (55 600) | 1.71 (25.0) | 0.625 (15.87) | 1.153 (29.29) |
| 100 | 1.250 (31.75) | 0.750 (19.05) | 19 500 (86 700) | 2.58 (37.7) | 0.750 (19.05) | 1.409 (35.76) |
| 120 | 1.500 (38.10) | 1.000 (25.40) | 28 000 (124 500) | 3.87 (56.5) | 0.875 (22.22) | 1.789 (45.44) |
| 140 | 1.750 (44.45) | 1.000 (25.40) | 38 000 (169 000) | 4.95 (72.2) | 1.000 (25.40) | 1.924 (48.87) |
| 160 | 2.000 (50.80) | 1.250 (31.75) | 50 000 (222 000) | 6.61 (96.5) | 1.125 (28.57) | 2.305 (58.55) |
| 180 | 2.250 (57.15) | 1.406 (35.71) | 63 000 (280 000) | 9.06 (132.2) | 1.406 (35.71) | 2.592 (65.84) |
| 200 | 2.500 (63.50) | 1.500 (38.10) | 78 000 (347 000) | 10.96 (159.9) | 1.562 (39.67) | 2.817 (71.55) |
| 240 | 3.00 (76.70) | 1.875 (47.63) | 112 000 (498 000) | 16.4 (239) | 1.875 (47.62) | 3.458 (87.83) |

► Table 17–19

Cadena y piñón



$$\sin \frac{\gamma}{2} \frac{p/2}{D/2} \quad \text{or} \quad D = \frac{p}{\sin(\gamma/2)}$$

$$D = \frac{p}{\sin(180^\circ/N)}$$

► Fig. 17-16 (a)

(17-29)

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Velocidad de cadena

$$V = \frac{Npn}{12} \text{ feet per minute} \quad (17 - 30)$$

- ▶ donde N = número de dientes de piñón
- ▶ p = paso de cadena, en mm
- ▶ n = velocidad de piñón, rev/min

$$v_{\max} = \frac{\pi Dn}{2} = \frac{\pi np}{12 \sin(\gamma/2)} \quad (b)$$

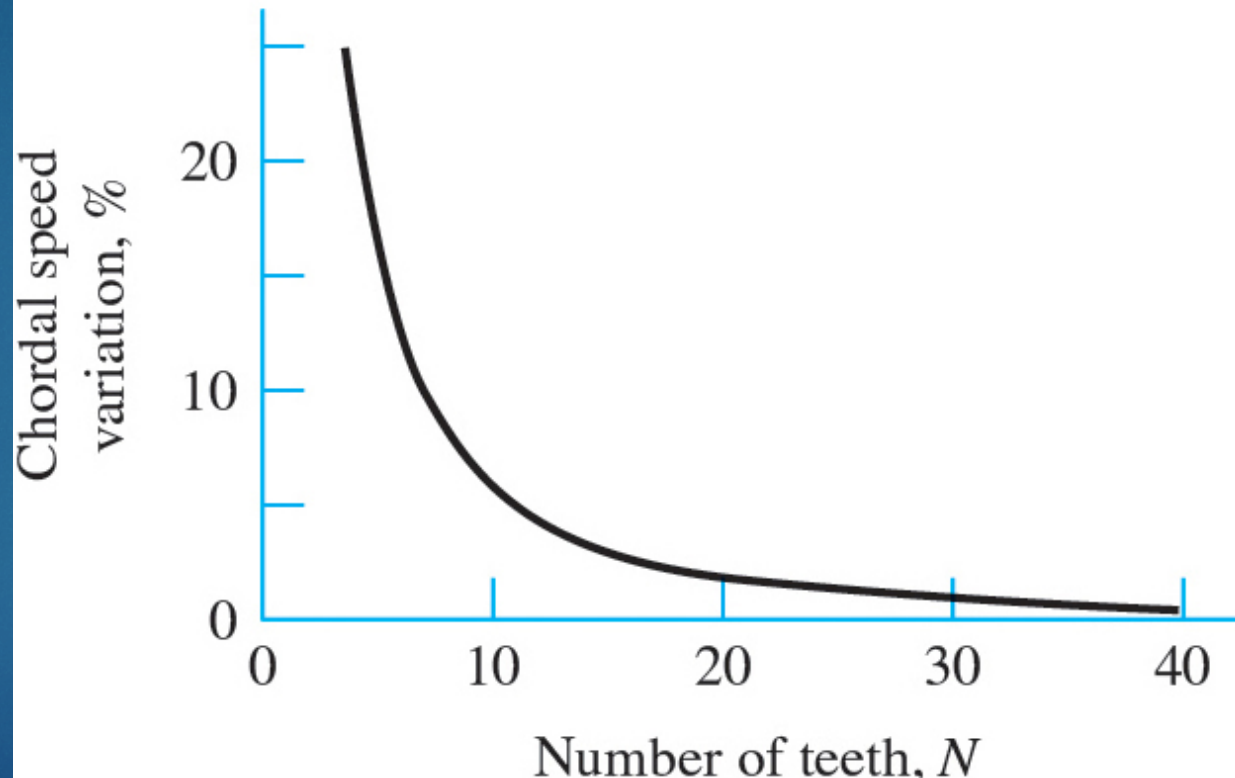
$$d = D \cos \frac{\gamma}{2} \quad (c)$$

$$v_{\min} = \frac{\pi dn}{12} = \frac{\pi np \cos(\gamma/2)}{12 \sin(\gamma/2)} \quad (d)$$

Variación de velocidad Cordal

$$\frac{\Delta V}{V} = \frac{v_{\max} - v_{\min}}{V} = \frac{\pi}{N} \left[\frac{1}{\sin(180^\circ/N)} - \frac{1}{\tan(180^\circ/N)} \right] \quad (17-31)$$

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► Fig.
17-18

Available Sprocket Tooth Counts

► **Table 17-21** Single-Strand Sprocket Tooth Counts Available from One Supplier*

| No. | Available Sprocket Tooth Counts |
|-----|--|
| 25 | 8-30, 32, 34, 35, 36, 40, 42, 45, 48, 54, 60, 64, 65, 70, 72, 76, 80, 84, 90, 95, 96, 102, 112, 120 |
| 35 | 4-45, 48, 52, 54, 60, 64, 65, 68, 70, 72, 76, 80, 84, 90, 95, 96, 102, 112, 120 |
| 41 | 6-60, 64, 65, 68, 70, 72, 76, 80, 84, 90, 95, 96, 102, 112, 120 |
| 40 | 8-60, 64, 65, 68, 70, 72, 76, 80, 84, 90, 95, 96, 102, 112, 120 |
| 50 | 8-60, 64, 65, 68, 70, 72, 76, 80, 84, 90, 95, 96, 102, 112, 120 |
| 60 | 8-60, 62, 63, 64, 65, 66, 67, 68, 70, 72, 76, 80, 84, 90, 95, 96, 102, 112, 120 |
| 80 | 8-60, 64, 65, 68, 70, 72, 76, 78, 80, 84, 90, 95, 96, 102, 112, 120 |
| 100 | 8-60, 64, 65, 67, 68, 70, 72, 74, 76, 80, 84, 90, 95, 96, 102, 112, 120 |
| 120 | 9-45, 46, 48, 50, 52, 54, 55, 57, 60, 64, 65, 67, 68, 70, 72, 76, 80, 84, 90, 96, 102, 112, 120 |
| 140 | 9-28, 30, 31, 32, 33, 34, 35, 36, 37, 39, 40, 42, 43, 45, 48, 54, 60, 64, 65, 68, 70, 72, 76, 80, 84, 96 |
| 160 | 8-30, 32-36, 38, 40, 45, 46, 50, 52, 53, 54, 56, 57, 60, 62, 63, 64, 65, 66, 68, 70, 72, 73, 80, 84, 96 |
| 180 | 13-25, 28, 35, 39, 40, 45, 54, 60 |
| 200 | 8-30, 32, 35, 36, 40, 42, 45, 48, 54, 56, 58, 59, 60, 63, 64, 65, 68, 70, 72 |
| 240 | 9-30, 32, 35, 36, 40, 44, 45, 48, 52, 54, 60 |

*Morse Chain Company, Ithaca, NY, Type B hub sprockets.

Tooth Correction Factors K_1

| Number of Teeth on Driving Sprocket | K_1 Pre-extreme Horsepower | K_1 Post-extreme Horsepower |
|-------------------------------------|------------------------------|-------------------------------|
| 11 | 0.62 | 0.52 |
| 12 | 0.69 | 0.59 |
| 13 | 0.75 | 0.67 |
| 14 | 0.81 | 0.75 |
| 15 | 0.87 | 0.83 |
| 16 | 0.94 | 0.91 |
| 17 | 1.00 | 1.00 |
| 18 | 1.06 | 1.09 |
| 19 | 1.13 | 1.18 |
| 20 | 1.19 | 1.28 |
| N | $(N_1/17)^{1.08}$ | $(N_1/17)^{1.5}$ |

► Table
17-22

Multiple-Strand Factors K_2

| Number of Strands | K_2 |
|-------------------|-------|
| 1 | 1.0 |
| 2 | 1.7 |
| 3 | 2.5 |
| 4 | 3.3 |
| 5 | 3.9 |
| 6 | 4.6 |
| 8 | 6.0 |

► Table
17-23

Clasificaciones nominales de potencia para cadena

- ▶ De la publicación de la Asociación Americana de Cadenas Cadenas Cadenas para la transmisión de energía y el manejo de materiales
- ▶ Para cadena de una sola hebra
- ▶ Potencia nominal, placa de enlace limitada
- ▶
$$H_1 = 0.004 N_1^{1.08} n_1^{0.9} p^{(3-0.07p)} \quad \text{hp} \quad (17-32)$$
- ▶ Potencia nominal, limitada por rodillos
- ▶
$$H_2 = \frac{1000 K_r N_1^{1.5} p^{0.8}}{n_1^{1.5}} \quad \text{hp} \quad (17-33)$$
- ▶ where N_1 = number of teeth in the smaller sprocket
 - ▶ n_1 = sprocket speed, rev/min
 - ▶ p = pitch of the chain, in
 - ▶ $K_r = 29$ for chain numbers 25, 35; 3.4 for chain 41; and 17 for chains 40–240

Chain Dimensions

- ▶ Chain length in pitches

$$\frac{L}{p} \approx \frac{2C}{p} + \frac{N_1 + N_2}{2} + \frac{(N_2 - N_1)^2}{4\pi^2 C/p} \quad (17-34)$$

- ▶ Center-to-center distance

$$C = \frac{P}{4} \left[-A + \sqrt{A^2 - 8 \left(\frac{N_2 - N_1}{2\pi} \right)^2} \right] \quad (17-35)$$

$$A = \frac{N_1 + N_2}{2} - \frac{L}{P} \quad (17-36)$$

Chain Drive Power

- ▶ Allowable power

$$H_a = K_1 K_2 H_{\text{tab}} \quad (17 - 37)$$

- ▶ where K_1 = correction factor for tooth number other than 17 (Table 17-22)
 - ▶ K_2 = strand correction (Table 17-23)
- ▶ Power that must be transmitted

$$H_d = H_{\text{nom}} K_s n_d \quad (17 - 38)$$

Variations in Tabulated Power Conditions

- ▶ Power ratings in Table 17–20 are for chains of 100 pitch length and 17-tooth sprocket.
- ▶ For deviations from this,

$$H_2 = 1000 \left[K_r \left(\frac{N_1}{n_1} \right)^{1.5} P^{0.8} \left(\frac{L_p}{100} \right)^{0.4} \left(\frac{15\,000}{h} \right)^{0.4} \right] \quad (17-39)$$

- ▶ From a deviation viewpoint,

$$\frac{H_2^{2.5} h}{N_1^{3.75} L_p} = \text{constant} \quad (17-40)$$

Recommended Maximum Chain Speed

$$n_1 \leq 1000 \left[\frac{82.5}{7.95^p (1.0278)^{N_1} (1.323)^{F/1000}} \right]^{1/(1.59 \log p + 1.873)} \quad \text{rev/min}$$

- ▶ where F is the chain tension in pounds

Chain Dimensions

- ▶ Chain length in pitches

$$\frac{L}{p} \approx \frac{2C}{p} + \frac{N_1 + N_2}{2} + \frac{(N_2 - N_1)^2}{4\pi^2 C/p} \quad (17-34)$$

- ▶ Center-to-center distance

$$C = \frac{P}{4} \left[-A + \sqrt{A^2 - 8 \left(\frac{N_2 - N_1}{2\pi} \right)^2} \right] \quad (17-35)$$

$$A = \frac{N_1 + N_2}{2} - \frac{L}{P} \quad (17-36)$$

Chain Drive Power

- ▶ Allowable power

$$H_a = K_1 K_2 H_{\text{tab}} \quad (17-37)$$

- ▶ where K_1 = correction factor for tooth number other than 17 (Table 17-22)
 - ▶ K_2 = strand correction (Table 17-23)
- ▶ Power that must be transmitted

$$H_d = H_{\text{nom}} K_s n_d \quad (17-38)$$

Variations in Tabulated Power Conditions

- ▶ Power ratings in Table 17–20 are for chains of 100 pitch length and 17-tooth sprocket.
- ▶ For deviations from this,

$$H_2 = 1000 \left[K_r \left(\frac{N_1}{n_1} \right)^{1.5} P^{0.8} \left(\frac{L_p}{100} \right)^{0.4} \left(\frac{15\,000}{h} \right)^{0.4} \right] \quad (17-39)$$

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- ▶ where F is the chain tension in pounds

Example 17-5₍₁₎

▶ Select drive components for a 2:1 reduction, 90-hp input at 300 rev/min, moderate shock, an abnormally long 18-hour day, poor lubrication, cold temperatures, dirty surroundings, short drive $C/p = 25$.

▶ **Solution**

▶ *Function:* $H_{\text{nom}} = 90 \text{ hp}$, $n_1 = 300 \text{ rev/min}$, $C/p = 25$

▶ *Design factor:* Choose $n_d = 1.5$

▶ *Service factor:* Choose $K_s = 1.3$ for moderate shock

▶ *Sprocket teeth:* $N_1 = 17$ teeth, $N_2 = 34$ teeth, $K_1 = 1$, $K_2 = 1, 1.7, 2.5, 3.3$

▶ *Chain number of strands:* From Equations (17-37) and (17-38), with $H_a = H_d$,

$$H_{\text{tab}} = \frac{n_d K_s H_{\text{nom}}}{K_1 K_2} = \frac{1.5(1.3)90}{(1)K_2} = \frac{176}{K_2}$$

Example 17–5₍₂₎

► Form a table:

| Número de Eslabones | 176/K2 (Table 17–23) | Chain Number (Table 17–20) | Lubrication Type |
|---------------------|-------------------------|-------------------------------|------------------|
| 1 | 176/1 = 176 | 200 | C' |
| 2 | 176/1.7 = 104 | 160 | C |
| 3 | 176/2.5 = 70.4 | 140 | B |
| 4 | 176/3.3 = 53.3 | 140 | B |

► **Decision**

► 3 strands of number 140 chain (from Table 17–20, H_{tab} is 72.4 hp).

► Number of pitches in the chain $\frac{L}{p} = \frac{2C}{p} + \frac{N_1 + N_2}{2} + \frac{(N_2 - N_1)^2}{4\pi^2 C/p}$

$$= 2(25) + \frac{17 + 34}{2} + \frac{(34 - 17)^2}{4\pi^2 (25)} = 75.79 \text{ pitches}$$

► **Decision**

► Use 76 pitches. Then $L/p = 76$.

Example 17-5₍₃₎

- Identify the center-to-center distance: From Equations (17-35) and (17-36),

$$A = \frac{N_1 + N_2}{2} - \frac{L}{p} = \frac{17 + 34}{2} - 76 = -50.5$$

$$C = \frac{p}{4} \left[-A + \sqrt{A^2 - 8 \left(\frac{N_2 - N_1}{2\pi} \right)^2} \right]$$
$$= \frac{p}{4} \left[50.5 + \sqrt{50.5^2 - 8 \left(\frac{34 - 17}{2\pi} \right)^2} \right] = 25.104p$$

- For a 140 chain, $p = 1.75$ in. Thus,

$$C = 25.104p = 25.104(1.75) = 43.93 \text{ in}$$

- Lubrication: Type B
- Comment: This is operating on the pre-extreme portion of the power, so durability estimates other than 15 000 h are not available. Given the poor operating conditions, life will be much shorter.