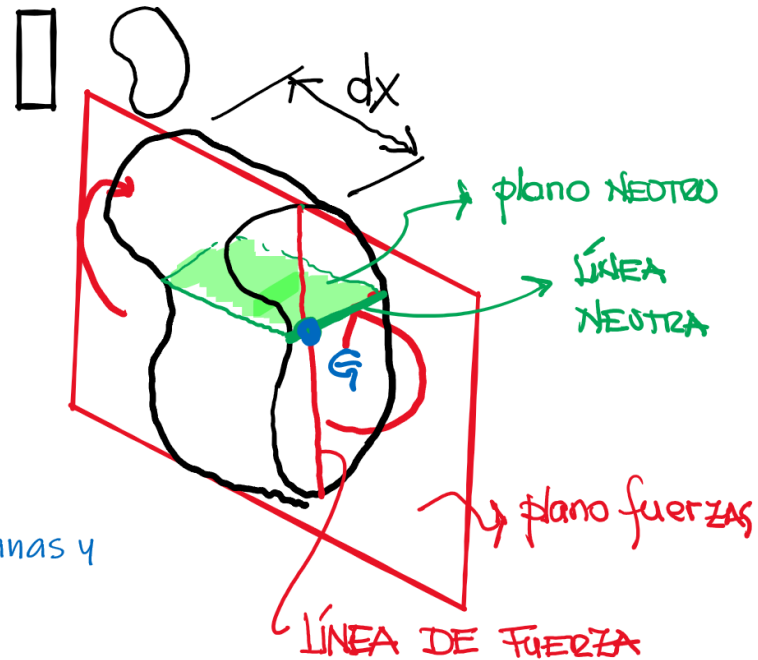
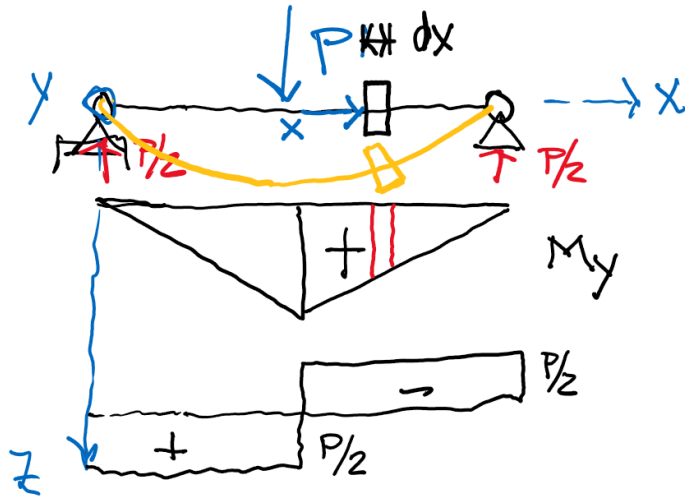
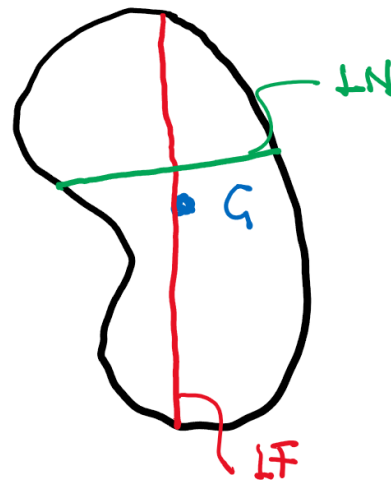
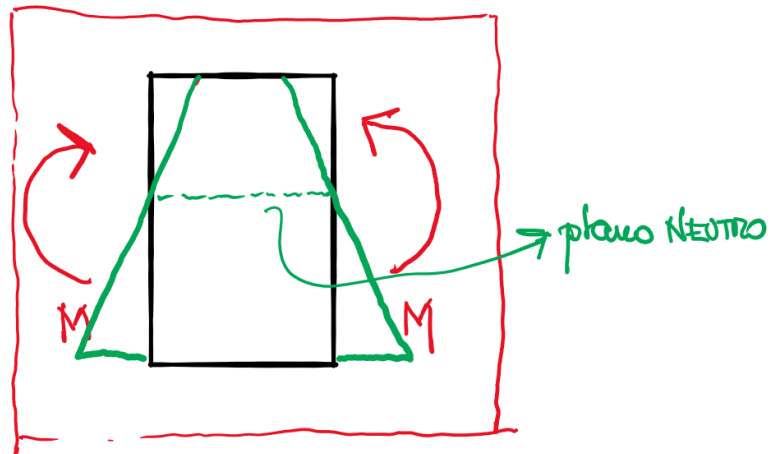


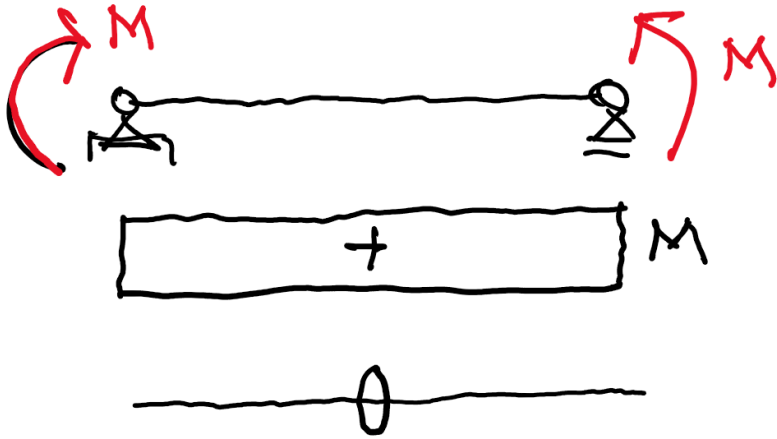
BARRAS SOLICITADAS A FLEXION



HIPOTESIS: Secciones se mantienen planas y perpendiculares al eje de barra



clasificación de flexión



Flexión constante

$$M \neq 0 \quad Q = 0$$

Flexión variable

$$M \neq 0 \quad Q \neq 0$$

Flexión Simple

$$M \neq 0 \quad N = 0$$

Flexión Compuesta

$$M \neq 0 \quad N \neq 0$$

Flexión RECTA

M COINCIDE CON EPI

Flexión OBLICUA

M NO COINCIDE CON EPI

Estudio de la deformación



$z \downarrow$

$$\epsilon_x = \frac{du}{dx}$$

$$\int \cdot d\theta_y = dx \rightarrow \underbrace{\left| \kappa_y = \frac{d\theta_y}{dx} \right|}_{\text{CURVATURA}} = \frac{1}{\rho}$$

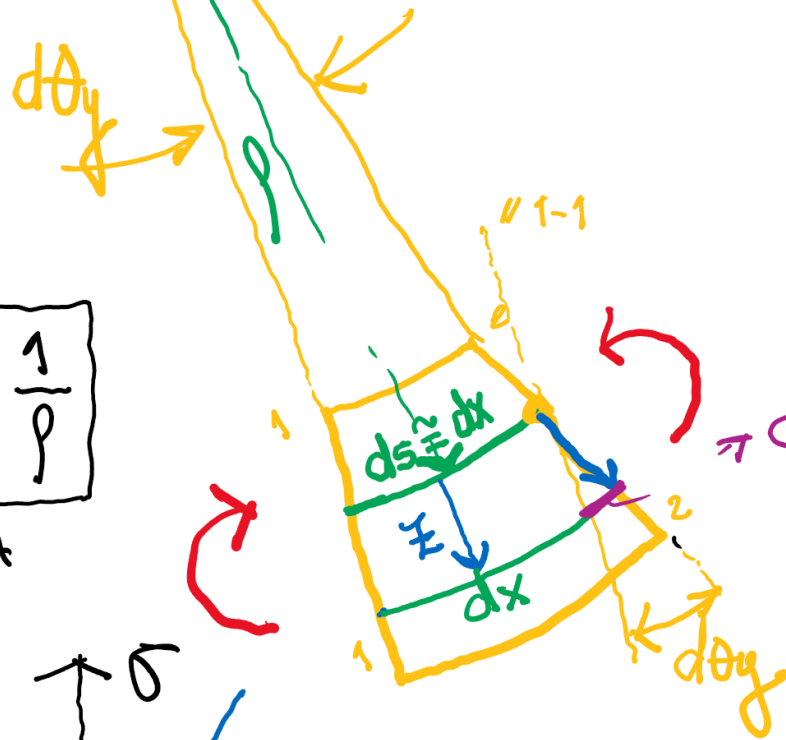
$$z \cdot d\theta_y = du = \epsilon_x \cdot dx$$

$$z \cdot \frac{d\theta_y}{dx} = \epsilon_x = \kappa_y \cdot z$$

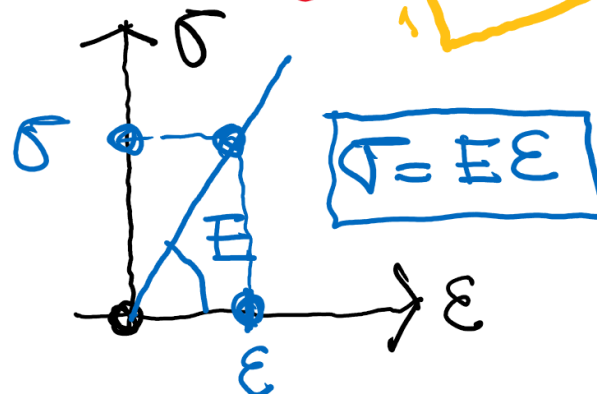
$$\sigma_x = E \cdot \kappa \cdot z$$

CENTRO DE LA CURVATURA

ρ = RADIO DE CURVATURA

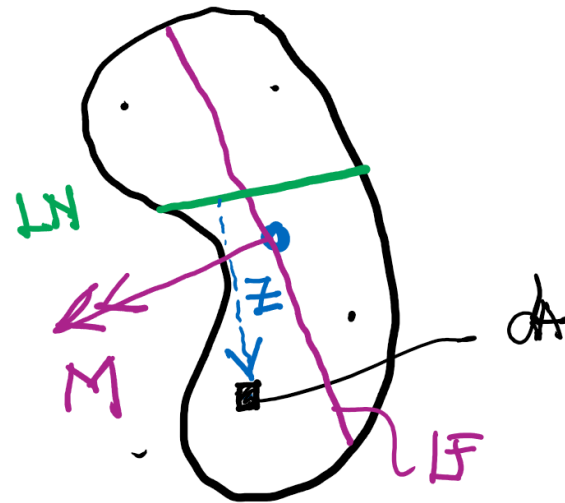
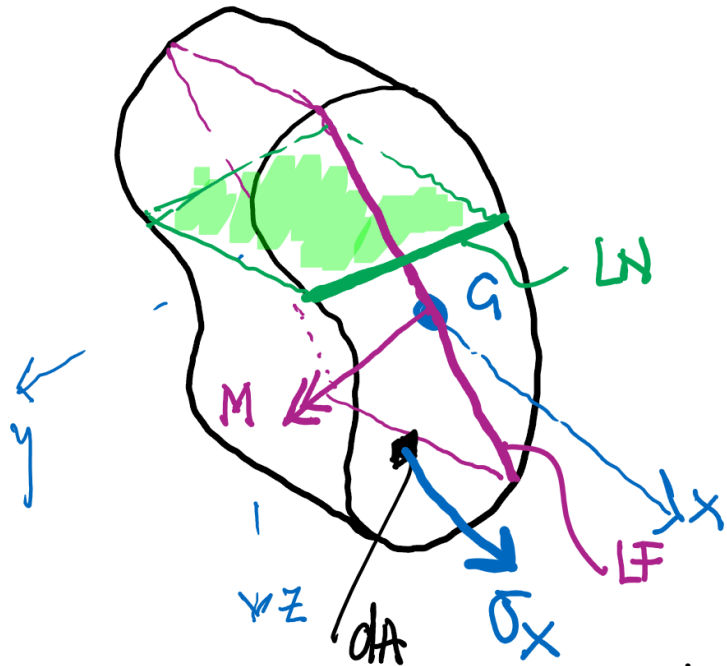


$$\rightarrow du = \epsilon_x \cdot dx$$



$$\sigma = E \epsilon$$

Ecuaciones de equivalencia

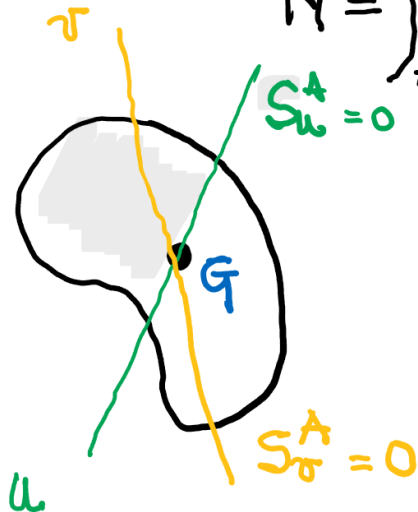


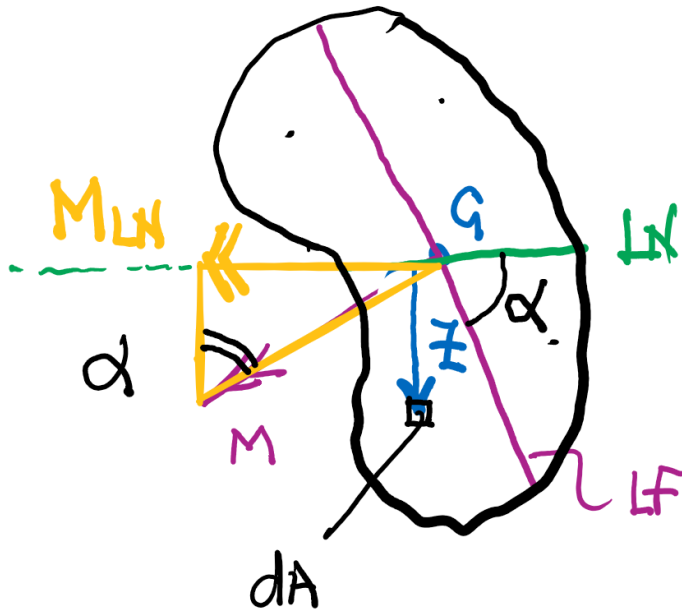
Flexión Simple

$$N = \int_A \sigma_x \cdot dA = \int_A E \cdot \chi_y \cdot z \cdot dA = E \cdot \chi_y \int_A z \cdot dA = 0$$

LN pasa x G

$$\Leftarrow S_{LN}^A = 0$$





$$M_{LN} = \int_A \sigma_x \cdot dA \cdot z = M \cdot \sin \alpha$$

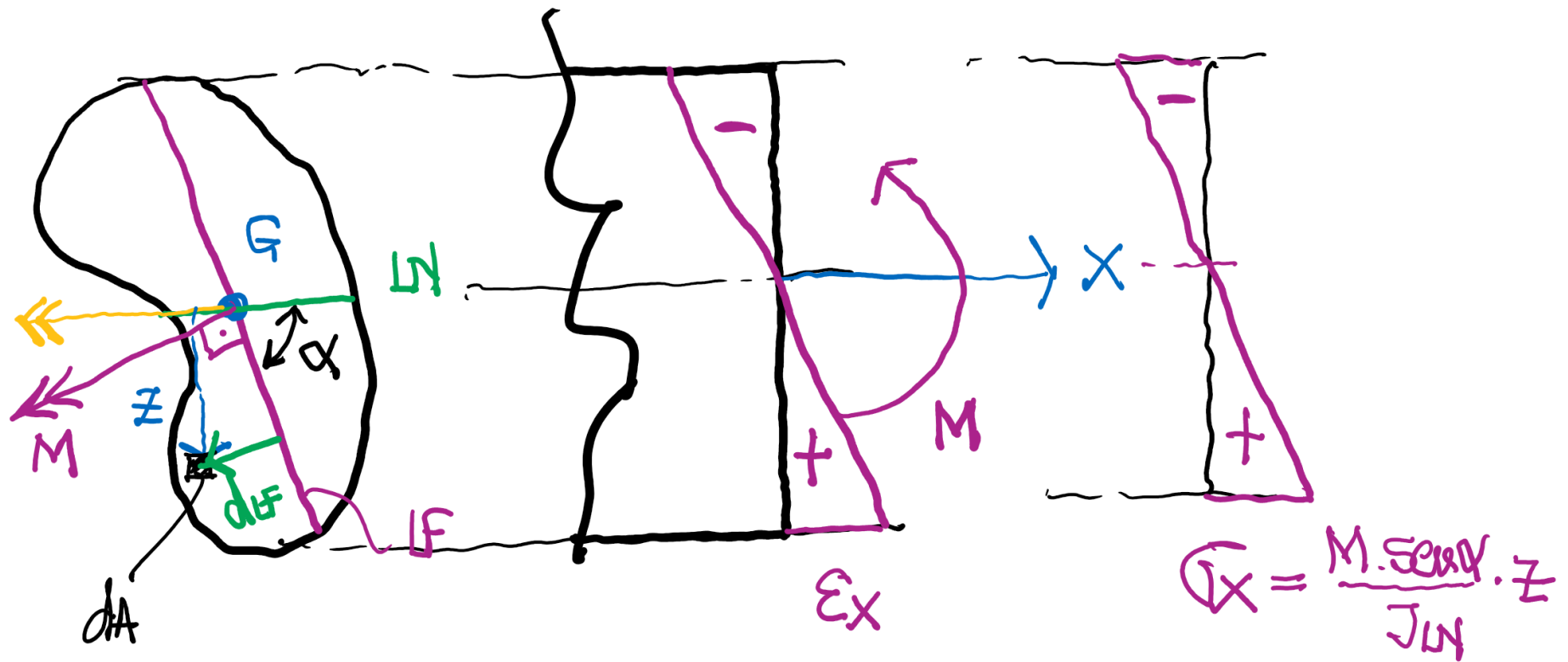
$$\alpha: \text{ang } e / LF_y \text{ LN}$$

$$M \sin \alpha = \int_A E \cdot \chi_y \cdot z^2 \cdot dA = E \chi_y \int_A z^2 dA$$

$$\chi_y = \frac{M \sin \alpha}{E \cdot J_{LN}} = \frac{d\theta_y}{dx}$$

J_{LN} ←

$$\sigma_x = \frac{M \cdot \sin \alpha}{J_{LN}} \cdot z$$



$$M_{LF} = \int_A \sigma_x \cdot dA \cdot dLF = \int_A \epsilon \cdot \chi_y \cdot z \cdot dLF \cdot dA = \epsilon \chi_y \int_A z \cdot dLF \cdot dA = 0$$

LN y LF SON EJES CONJUGADOS $\leftarrow J_{LN;LF} = 0$